

ENGLISH

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CENTAX-K AT A GLANCE

A coupling for diesel-driven compressor or pump drives should have one primary feature: to be as uncompromising regarding ease in mounting as our new CENTAX-K.

The torsionally soft coupling is based on a rubber element that features high torsional flexibility and radial movability vulcanised to a glass-fibre reinforced plastic flange with an integrated hub on the output side of the KS design and a seperate hub star for the KK design. The coupling designs can be plugged in axially to ensure maximum ease in mounting. And, the design is also impressive considering its dimensions and cost.

The torsional flexibility of the CENTAX-KS dampens torsional vibrations and impacts and compensates for any axial and radial misalignments due to operation. The rubber element for this coupling is available in various designs. Currently, the series covers torques ranging from 400 to 800 Nm.

Features

High torsional flexibility Compact dimensions Maximum ease in mounting Low-cost

Areas of application



Compressor drives Pump drives Hydrostatic drives

Torque range

400 – 800 Nm Higher torques upon request

For efficient torque transmission

LEADING O BY INNOVATION





Reliable power transmission

for your applications with an

optimum of features.



The CENTAX rubber elements are available in different degrees of Shore hardness. This enables the torsional flexibility of the couplings to be adapted with utmost variability to the specific application. Torsional vibrations and impacts are reliably dampened. The CENTA coupling programm offers many functions to protect your drive from harmful torsional vibrations, to compensate misalignment and to dampen vibrations and noise.



MISALIGNMENT CAPABILITY

CENTA's coupling systems convince with kinematics unique to the market. They offer the necessary misalignment capability for each application, maximized through numeric calculations and endurance tests (type test). Ensuring a reliable compensation of misalignments even with short drive shafts and an efficient damping of vibration and noise.



MODULARITY

All standard flywheel connections can be realized with our CENTAX-K allowing for adequate design for any application. For efficient and fast customized solutions.



QUALITY

When the going gets tough, quality is priceless. With an exemplary Quality Management, CENTA ensures products that withstand the roughest assignments. CENTA's coupling systems are more than the sum of their parts. CENTA entertains the vision of intelligent products that meet the highest requirements in terms of design and quality.

DESIGN TYPES

Which product for your purpose? We will gladly assist → www.centa.info/contact

CENTAX-K DESIGN TYPES



CENTAX-KK

The CENTAX-KK is a two piece connection to the driven units. By use of the additional well proven hubstar of CENTAFLEX-K an axial plug-in is achieved. The design thus allows for key connections, conical clamping or the CENTALOC-clamping. The positive locking of the hubstar by an integrated rubber bumper resulted in a further innovation. The fix rotating direction coupling becomes free of backlash and is noise reduced during operation.



CENTAX-KN

The CENTAX-KN is a two piece connection to the driven units. By use of the additional well proven hubstar of the CENTAFLEX-K in steel or aluminium, an axial plug-in is achieved. The design thus allows for key connections, conical clamping or the CENTALOC-clamping.



CENTAX-KS

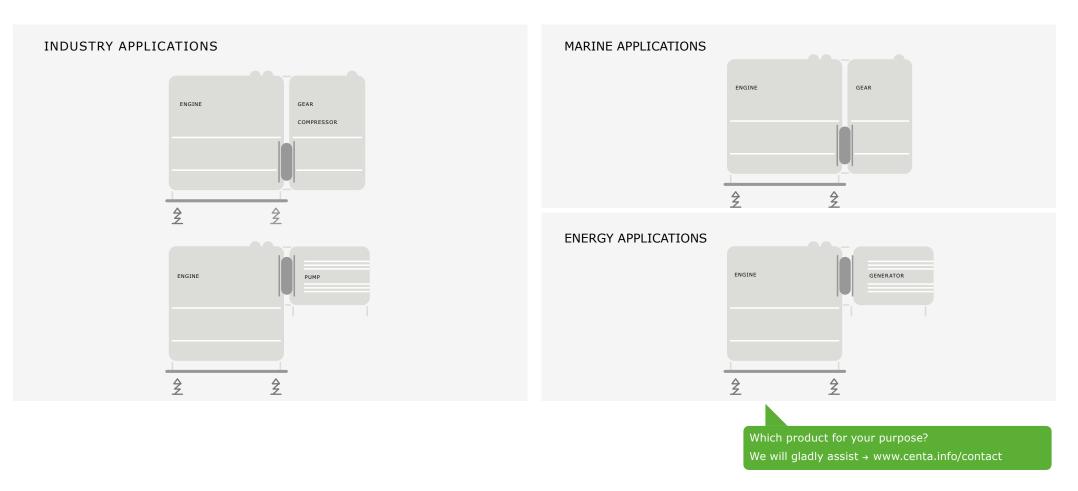
The hub of CENTAX-KS is integrated into the plastic flange on the driven side.

The driving unit is plugged onto the customer-specific spline of this one-piece coupling.

APPLICATIONS

Which product for your purpose? We will gladly assist → www.centa.info/contact

CENTAX-K APPLICATIONS



TECHNICAL DATA

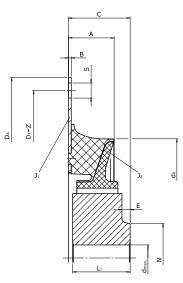
TECHNICAL DATA	DIMENSIONS
CENTAX-KK/KS	CENTAX-KK
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CENTAX-KN	CENTAX-KS
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	CENTAX-KN
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Explanation of the table headers \rightarrow page APP-1

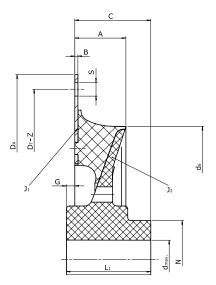
CENTAX-KK/KS

TECH	INICAL DATA		↓ SIZES 22-	-28									
1	2	3	4	5	6*	7*	8	9	10	12	13	14	
Size	Rubber quality	Nominal torque	Maximum torque			Dynamic torsional stiffness	Relative damping	Speed	Permissible axial displacement	Permissible radial displacement	Radial stiffness	Permissible angular displacement	Flange size
		Тки	Tĸmax	Ткw	Ρκν	CTdyn	Ψ	n _{max}	ΔKa	ΔKr	Cr	ΔKw	SAE
	[Shore A]	[kNm]	[kNm]	[kNm]	[W]	[kNm/rad]		[min⁻¹]	[mm]	[mm]	[kN/mm]	[°]	J620
22	50	0,40	0,80	0,16	150	2,6	1,05	5000	±2	0,6	0,75	0,5	7,5 8 10
22	60	0,50	1,00	0,20	150	3,2	1,10	5000	12	0,0	0,85	0,5	7,5 8 10
25	50	0,55	1,10	0,22	100	3,5	1,05	5000	1.2	0.0	1,00	0,5	0 10
25	60	0,68	1,36	0,27	160	4,3	1,10	5000	±2	0,6	1,10	0,5	8 10
20	50	0,63	1,26	0,25	170	4,0	1,05	5000	1.2	0.0	1,10	0,5	10 11 5
28	60	0,80	1,60	0,32	170	5,0	1,10	5000	±2	0,6	1,20	0,5	10 11,5

* preliminary values



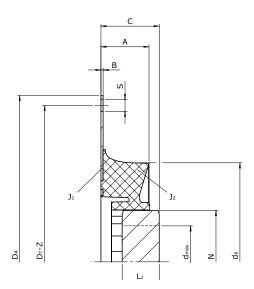
DIM	ENSIONS		↓ s:	IZES 22-	28														
Size	Nominal torque		Dimensions							Flange dimensions					Mass moments of intertia and masses				
	Тки	А	В	С	d _{ma x}	de	E	L1	N	SAE	DA	Dτ	Z	S	J_1	J ₂	m1	m2	m _{total}
	[kNm]									J620	[-0,15]	[±0,2]			[kg	m²]		[kg]	
	1					[1					1	1	1				
										7,5	241,3	222,3	8x45°	9	0,006	0,003	0,74	1,56	2,30
22	0,40 - 0,50	33,3	2	45	40	174	6	42	50	8	263,5	244,5	6x60°	11	0,008	0,003	0,88	1,56	2,44
										10	314,3	295,3	8x45°	11	0,015	0,003	1,23	1,56	2,79
25		35,4	C	45	40	187	6	42	50	8	263,5	244,5	6x60°	11	0,008	0,004	0,90	1,69	2,59
25	0,55 - 0,68	35,4	2	45	40	107	6	42	50	10	314,3	295,3	8x45°	11	0,016	0,004	1,26	1,69	2,95
28	0.62 0.90	36,6	2	45	40	194	6	42	50	10	314,3	295,3	8x45°	11	0,016	0,004	1,30	1,74	3,04
28	0,63 - 0,80	0,02	2	45	40	194	0	42	50	11,5	352,4	333,4	8x45°	11	0,025	0,004	1,62	1,74	3,36



DIM	ENSIONS		↓ s:	IZES 22-	28														
Size	Nominal torque		Dimensions							Flange dimensions					Mass moments of intertia and masses				
	Тки	А	В	С	d _{max}	d6	G	L1	Ν	SAE	DA	Dτ	Z	S	J_1	J ₂	m1	m ₂	m total
	[kNm]									J620	[-0,15]	[±0,2]			[kg	m²]		[kg]	
	1											1		1					
										7,5	241,3	222,3	8x45°	9	0,006	0,002	0,71	0,56	1,27
22	0,40 - 0,50	33,3	2	49,5	40	174	5,5	55	51	8	263,5	244,5	6x60°	11	0,008	0,002	0,84	0,56	1,40
										10	314,3	295,3	8x45°	11	0,015	0,002	1,19	0,56	1,75
25		35,4	2	49,5	40	187	5,5	55	51	8	263,5	244,5	6x60°	11	0,009	0,003	0,91	0,65	1,56
25	0,55 – 0,68	35,4	2	49,5	40	107	5,5	55	51	10	314,3	295,3	8x45°	11	0,016	0,003	1,26	0,65	1,91
28	0.62 0.90	26.6	2	49,5	40	194		55	51	10	314,3	295,3	8x45°	11	0,016	0,003	1,25	0,76	2,01
20	0,63 - 0,80	36,6	2	49,5	40	194	5,5	55	51	11,5	352,4	333,4	8x45°	11	0,025	0,003	1,56	0,76	2,32

TECH	INICAL DATA		↓ SIZES 22-	-28										
1	2	3	4	5	6*	7*	8	9	10	12	13	14		
Size	Rubber quality	Nominal torque	Maximum torque	Continuous Permissit vibratory power lo torque		Dynamic torsional stiffness	Relative damping	Speed	Permissible axial displacement	Permissible radial displacement	Radial stiffness	Permissible angular displacement	Flange size	
		Тки	Tĸmax	Ткw	Ρκν	CTdyn	Ψ	n _{max}	ΔKa	ΔKr	Cr	ΔKw	SAE	
	[Shore A]	[kNm]	[kNm]	[kNm]	[W]	[kNm/rad]		[min ⁻¹]	[mm]	[mm]	[kN/mm]	[°]	J620	
22	50	0,40	0,80	0,16	150	1,80	1,05	5000	±2	0,6	0,75	0,5	7,5 - 8 - 10	
22	60	0,50	1,00	0,20	150	2,70	1,10	5000	12	0,0	0,85	0,5	/,5 - δ - 10	
25	50	0,55	1,10	0,22	100	2,40	1,05	5000	±2	0.0	1,00	0,5	0 10	
25	60	0,68	1,36	0,27	160	3,60	1,10	5000	=	0,6	1,10	0,5	8 - 10	
20	50	0,63	1,26	0,25	170	3,00	1,05	5000		0.6	1,10	0,5	10 11 5	
28	60	0,80	1,60	0,32	170	4,50	1,10 5000		±2	0,6	1,20	0,5	10 - 11,5	

* preliminary values



DIM	IENSIONS																		
Size	Nominal torque	l torque Dimensions						Flange dimensi						М	Mass moments of intertia and masses				
	Tĸℕ [kNm]	А	В	С	d _{max}	d₅	Lı	N	SAE J620	D₄ [-0,15]	D⊤ [±0,2]	Z	S	Jı [kg	J ₂ Jm ²]	m1	m₂ [kg]	M total	
															-				
									7,5	241,3	222,3	8x45°	9	0,007	0,004	0,76	2,32	3,08	
22	0,40 - 0,50	43,4	2	55	68	174	35	97	8	263,5	244,5	6x60°	11	0,009	0,004	0,90	2,32	3,22	
									10	314,3	295,3	8x45°	11	0,016	0,004	1,25	2,32	3,57	
25		45.4	2	55	68	187	35	97	8	263,5	244,5	6x60°	11	0,010	0,005	0,99	2,44	3,43	
25	0,55 - 0,68	45,4	2	55	68	187	35	97	10	314,3	295,3	8x45°	11	0,017	0,005	1,34	2,44	3,78	
20	0.62 0.00	AC 4	2	FF	60	104	25	07	10	314,3	295,3	8x45°	11	0,017	0,006	1,40	2,51	3,91	
28	0,63 - 0,80	46,4	2	55	68	194	35	97	11,5	352,4	333,4	8x45°	11	0,026	0,006	1,71	2,51	4,22	

EXPLANATION OF THE TECHNICAL DATA

This appendix shows all explanations of the technical data for all CENTA products.

the green marked explanations are relevant for this catalog:

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2	Rubber quality	Page APP-2
3		Page APP-2
4		Page APP-2
5	Continuous vibratory torque	Page APP-2
6	Permissible power loss	Page APP-2
7	Dynamic torsional stiffness	Page APP-3
8		Page APP-3
9	Speed	Page APP-3
10	Permissible axial displacement	Page APP-3
	Axial stiffness	Page APP-4
12	Permissible radial displacement	Page APP-4
13	Radial stiffness	Page APP-4
14	Permissible angular displacement	Page APP-4
	Angular stiffness	Page APP-4

Are these technical explanations up to date? click here for an update check!

EXPLANATION OF THE TECHNICAL DATA

1	
Size	

This spontaneously selected figure de ignates the size of the coupling.

2		4					
Rubber quality		Maximum torque					
Shore A	[kNm]						
This figure indicates the nominal shore hardness of the elastic element. The nominal value and the effective val- ue may deviate within given tolerance ranges.	Tĸmax	This is the torque that may occur occasionally and for a short period up to 1.000 times and may not lead to a substantial temperature rise in the rubber element.					
3	In additi	on the following maximum tor-					
Nominal torque	ques may occur:						
Τ _{κν} [kNm]	ΔT _{Kmax} =	Peak torque range (peak to peak) between maximum and					
verage torque which can be transmit- ed continuously over the entire speed	1,8xTĸN	minimum torque, e.g. switch- ing operation.					
range	Т _{ктах1} = 1,5хТкм	Temporary peak torque (e.g. passing through resonances). ΔT_{Kmax} or T_{Kmax1} may occur 50.000 times alternating or 100.000 times swelling.					
	T _{Kmax2} = 4,5xT _{KN}	Transient torque rating for very rare, extraordinary con- ditions (e.g. short circuits).					

St PKY 0,8 0,6 0,4 0,2 20 30 40 50 60 70 80 90

Continuous vibratory torque T_{KW} [kNm]

Amplitude of the continuously permissible periodic torque fluctuation with a basic load up to the value T_{KN} .

1,0

The frequency of the amplitude has no influence on the permissible continuous vibratory torque. Its main influence on the coupling temperature is taken into consideration in the calculation of the power loss.

Operating torque

T_{Bmax} [kNm]

The maximum operating torque results of TKN and TKW.

Permissible Power Loss P_{κν} [kW] or [W]

Damping of vibrations and displacement results in power loss within the rubber element.

The permissible power loss is the maximum heat (converted damping work into heat), which the rubber element can dissipate continuously to the environment (i.e. without time limit) without the maximum permissible temperature being exceeded.

The given permissible power loss refers to an ambient temperature of 30° C. If the coupling is to be operated at a higher ambient temperature, the temperature factor StPKV has to be taken into consideration in the calculation.

The coupling can momentarily withstand an increase of the permissible power loss for a short period under certain operation modes (e.g. misfiring).

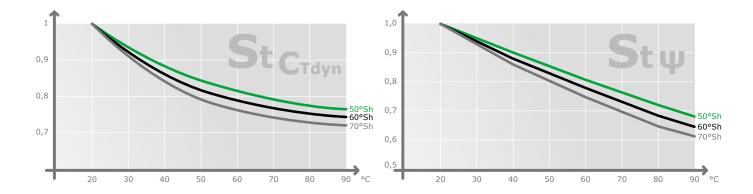
Ркv30 [kW] or [W]

For a maximum period of 30 minutes the double power loss PKV30 is permissible. CENTA keeps record of exact parameters for further operation modes.

• Temperature

• Frequency of vibration

• Amplitude of vibratory torque



EXPLANATION OF THE TECHNICAL DATA

Higher temperature reduces the dynamic torsional stiffness.

Higher frequencies increase the torsional stiffness.

ness. CENTA keeps record of exact parameters.

Temperature factor $S_t\,c_{Tdyn}$ has to be taken into consideration in the calculation.

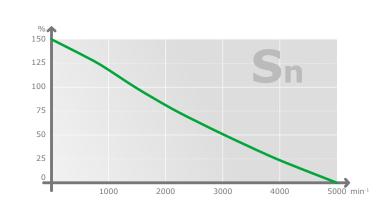
By experience the dynamic torsional stiffness is 30% higher than the static stiff-

Higher amplitudes reduce the torsional stiffness, therefore small amplitudes result

in higher dynamic stiffness. CENTA keeps record of exact parameters.

7	8	9	10			
Dynamic torsional stiffness	Relative damping	Speed	Permissible axial displacement [mm]			
C _{Tdyn} [kNm/rad]	Ψ	[min ⁻¹]				
The dynamic torsional stiffness is the relation of the torque to the torsional angle under dynamic loading. The torsional stiffness may be linear or progressive depending on the coupling design and material. The value given for couplings with linear torsional stiffness considers following terms:	The relative damping is the relationship of the damping work to the elastic de- formation during a cycle of vibration. The larger this value $[\Psi]$, the lower is the increase of the continuous vibratory torque within or close to resonance.	The maximum speed of the cou- pling element, which may occur occasionally and for a short pe- riod (e.g. overspeed). Nmax The characteristics of mounted parts may require a reduction of	 The continuous permissible axial displacement of the coupling. ΔK_a This is the sum of displacement by assembly as well as static and dynamic displacements during operation. 			
• Pre-load: 50% of TKN • Amplitude of vibratory torque: 25% of TKN • Ambient temperature: 20°C • Frequency: 10 Hz	The tolerance of the relative damping is ±20%, if not otherwise stated. The relative damping is reduced at higher temperatures.	the maximum speed (e.g. outer diameter or material of brake discs). The maximum permissible	The maximum axial displace- ment of the coupling, which may occur occasionally for a short			
For couplings with progressive torsional stiffness only the pre-load value changes as stated. The tolerance of the torsional stiffness is $\pm 15\%$ if not stated otherwise.	Temperature factor $S_{t^{\Psi}}$ has to be taken into consideration in the calculation. The vibration amplitude and frequency only have marginal effect on the rela- tive damping.	nd speed of highly flexible cou- pling elements is normally 90% thereof.	is handled in technical docu- ments (displacement diagrams, data sheets, assembly instruc-			
The following influences need to be considered if the torsional stiffness is required for other operating modes:			tions).			

EXPLANATION OF THE TECHNICAL DATA



	11		12		13		14	15		
	Axial stiffness	P	ermissible radial displacement		Radial stiffness	Pe	ermissible angular displacement		Angular stiffness	
	[kN/mm]	[mm]			[kN/mm]		[} ⁰]		[kNm/°]	
Ca	The axial stiffness determines the axial reaction force on the input and output sides upon axial displacement.		The continuous permissible radi- al displacement of the coupling. This is the sum of displacement by assembly as well as static and dynamic displacements dur-	Cr	The radial stiffness determines the radial reaction force on the input and output sides upon ra- dial displacement.		The continuous permissible an- gular displacement of the cou- pling. This is the sum of displacement	Cw	The angular stiffness determines the restoring bending moment on the input and output sides upon angular displacement.	
Ca dyn	By experience the dynamic stiff- ness is higher than the static one. The factor depends on the coupling series.	ΔKr	a ∆Kr ir d tl g S	ΔK_r ing operation. The continuous permissible ra- dial displacement depends on the operation speed and may re- quire adjustment (see diagrams S_n of the coupling series).		By experience the dynamic stiff- ness is higher than the static one. The factor depends on the coupling series.	ΔKw	by assembly as well as static and dynamic displacements dur- ing operation. The continuous permissible an- gular displacement depends on the operation speed and may re- quire adjustment (see diagrams S_n of the coupling series).	Cwdyn	By experience the dynamic stiff- ness is higher than the static one. The factor depends on the coupling series.
	Δ	∆Kr ma	The maximum radial displace- ment of the coupling, which may occur occasionally and for a short period without considera- tion of the operation speed (e.g. extreme overload). ⁵ The concurrent occurrence of different kinds of displacements is handled in technical docu- ments (displacement diagrams, data sheets, assembly instruc- tions).			ΔKwm	The maximum angular displace- ment of the coupling, which may occur occasionally and for a short period without considera- tion of the operation speed (e.g. extreme overload). The concurrent occurrence of different kinds of displacements is handled in technical docu- ments (displacement diagrams, data sheets, assembly instruc- tions).			

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1. This catalog supersedes previous editions.

This catalog shows the extent of our CENTAX®-SEC coupling range at the time of printing. This program is still being extended with further sizes and series. Any changes due to technological progress are reserved.

We reserve the right to amend any dimensions or detail specified or illustrated in this publication without notice and without incurring any obligation to provide such modification to such couplings previously delivered. Please ask for an application drawing and current data before making a detailed coupling selection.

2. We would like to draw your attention to the need of preventing accidents or injury. No safety guards are included in our supply.

3. TRADEMARKS

CENTA, the CENTA logo, Centacone, CENTADISC, CENTAFIT, Centaflex, CENTALINK, Centalock, Centaloc, Centamax, Centastart, CENTAX and HYFLEX are registered trademarks of CENTA Antriebe Kirschey GmbH in Germany and other countries.

Other product and company names mentioned herein may be trademarks of their respective companies.

4. Torsional responsibility

The responsibility for ensuring the torsional vibration compatibility of the complete drive train, rests with the final assembler. As a component supplier CENTA is not responsible for such calculations, and cannot accept any liability for gear noise/-damage or coupling damage caused by torsional vibrations.

CENTA recommends that a torsional vibration analysis (TVA) is carried out on the complete drive train prior to start up of the machinery. In general torsional vibration analysis can be undertaken by engine manufacturers, consultants or classicfication societies. CENTA can assist with such calculations using broad experience in coupling applications and torsional vibration analysis.

- 5. Copyright to this technical dokument is held by CENTA Antriebe Kirschey GmbH.
- 6. The dimensions on the flywheel side of the couplings are based on the specifications given by the purchaser. The responsibility for ensuring dimensional compatibility rests with the assembler of the drive train. CENTA cannot accept liability for interference between the coupling and the flywheel or gearbox or for damage caused by such interference.
- 7. All technical data in this catalog are according to the metric SI system. All dimensions are in mm. All hub dimensions (N, N1 and N2) may vary, depending on the required finished bore. All dimensions for masses (m), inertias (J) and centres of gravity (S) refer to the maximum bore diameter.





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